

Research Article

Optimization Technique Focused on Back-Pressure Production Occurrences of Fixed 4-Stroke Diesel Generator using ANN & DA Modeling

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Abstract

This paper focuses on the phenomena of backpressure mostly in context of a compression ignition engine. Efficient use of treatment strategies explicitly with C.I. engine needs a crucial examination of the overall exhaust system used. Searching for Diesel Particulate Filters mostly as technological advances is indeed very effective since particulates are known as a common cancer source. Backpressure primarily functioning upon this engine is perhaps the critical aspect that essentially performance of that same engine is significantly impacted as well as air pollution system requirements. The current study simulation is done to determine the correlation seen between geometrical parametric variations of its exhaust component of the system for assessment of a backpressure generation phenomenon. Backpressure was created on a static C.I. Engine by attaching a new diesel particulate filter to the exhaust system for the test case. As per methodology suggested for design of the experiments, testing of the prepared setup is done. That value of the coefficient of correlation between both the data observed and the data sets computed is calculated. The research finding shows an essential context for improving it's designed to operate efficiency of the system by optimizing the model for all types of I.C. Engines.

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KEYWORDS: Diesel engine, DPF, Backpressure generation, DA Modeling, ANN, Performance Analysis.

1. Introduction

Amongst all of the types of I. C. engines recognized today, that diesel engine is perhaps the most high energy efficient engine. Such high performance converts into better fuel efficiency and low emissions of greenhouse gases. Durability, performance, including energy conservation offer alternative diesel specifications which haven't been challenged by rival energy conversion equipment. Diesel drawbacks include vibration, low specific power generation, pollution of NO_x and PM, and increased price [1].

That is a complicated structure that consists of several other highly complicated system components, separately with another regarding the automotive power plant mostly as entire solution, and they mostly share

certain similar characteristics and objectives that enable them to function together. To obtain the maximum conversion of fuel energy with lowest vehicle tailpipe emissions, total systems approach is required, objectives that enable them to function together. To obtain the maximum conversion of fuel energy with lowest vehicle tailpipe emissions, total systems approach is required, where the design of

the engine, the control system and exhaust gas aftertreatment should be balanced. Engine aftertreatment requirement changes in response to changing pollution control norms or legislation, this typically includes the implementation of new technology [1].

Emerging innovations bring unique. Challenges to post-treatment solutions. A brief overview regarding the main trends in aftertreatment system development and problems associated are given here. After successful engine system design also backpressure on a particular engine increases because of following reasons: Installation of additional device such as Trap, catalytic converter, EGR system, Turbocharger etc., if after new inventions that are to be implemented. Engine operating condition such as load and speed decides the fuel combustion and ultimately backpressure for a particular engine system. Majority of the C.I. Engine technologies limitation of room availability requires sturdiness of after-treatment systems, creates restrictions on exhaust flow. Due servicing including adjustment of all parts seems to be a very important consideration for maintenance free, effective as well as productive life of its engine system. Each type of fuel or lubrication oil variations for a particular engine system decides the engine operation and ultimately backpressure on engine system. Its most influential diesel particulates pollution control technique enables the particulate filtration system. Collected particles were extracted from its device, either consistently or regularly, by thermal regeneration. The system receives ash, however the aggregation of ash throughout the trap becomes large enough to cause an increase in back pressure. Thus backpressure on engine, which depends on all possible system design and operating conditions (such as type of engine, fuel, lubrication oil and exhaust system) is the deciding factor for effectiveness of aftertreatment system design and finally overall system performance [1, 2, 3].

Even as exhaust system architects also encountered back pressure constraints, a significant rise in exhaust pressure was also caused by diesel engine after treatment devices, like diesel particulate filters (DPF), but also by the implementation of specific after-treatment systems. Installing DPFs also expressed concern regarding intensified exhaust back pressure. But so much of the exhaust gas pressure drop over its DPF appears to be caused mostly due to the deposit of soot instead of the filter membrane. Issues start when the regeneration, including its DPF doesn't really happen on either a constant schedule as well as its pressure decreases to the an inappropriate amount.

Elevated exhaust pressure could have a range of symptoms mostly on diesel engine which are as follows:

- Improved pumping effort;
- Lowered intake by multiple boosting pressure;
- Scavenging, including its effect on oxidation inside cylinders;
- Turbocharger concerns;

Many engines get the maximum permissible back pressure of the engine indicated mostly by engine supplier. Functioning that engine with high back pressure could disprove the warranty of that same engine. In order to promote its retrofit for installed DPF engines, to particular through using passive DPF systems, pollution control suppliers as well as engine consumers have demanded that automotive companies raise the overall permissible back pressure levels on certain engines [1, 5, 6, 7]

2. Concerns of Backpressure on C. I. Engine

Within that scenario, the goals including its engine system are also to improve fuel economy and reduce emission levels. The significant element for improving engine efficiency has been the design of its exhaust system including minimal back pressure requirements for such goals. As the indicator diagram shows the work done per cycle and the backpressure rise during exhaust process is increasing the negative area of the diagram as shown by section lines above the atmospheric pressure during exhaust stroke refer figure -1. During experimentation average increase in the value of backpressure (P_b) is equal to 0.73 % at the input, the average increase in the value of fuel consumed (F) is observed is equal to 83.25 % at the output. It clearly indicates that a small increase in the value of backpressure drastically increases the fuel consumed

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[8, 9]

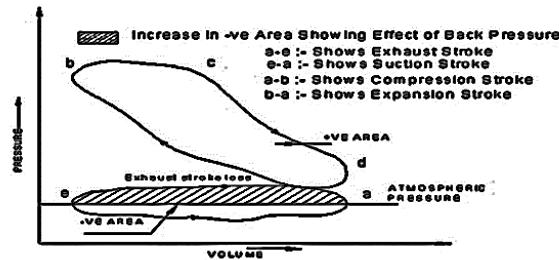


Figure.01: Explanation of the concept of backpressure using indicator diagram

A complete knowledge of the back-pressure impact on diesel engine output and to define appropriate back-pressure limitations together with measures to solve high back-pressure, the pulse turbocharged, medium speed, diesel engine can be monitored at various loads as well as engine speeds, with distinct characteristics for static back pressure. In specific, its mean value model measurements might be tested which was used to test both the pulse output as well as the constant pressure turbocharged engine for both the high back pressure of 1 metre water column (mWC) including 2 different valve overlap values. The mathematical justification for engine smoke threshold and other thermal overloading were explored through using established simulation model. The approach applying the philosophic rationale for determining the acceptable back pressure limits has also been taken into account. The integration with pulse turbocharger systems as well as slight valve overlapping demonstrated a major increase in the back pressure handling capabilities of the engines [3]

Some analysis incorporates the observed results, including GT-SUITE simulations, to assess the efficiency of exhaust back-pressure (P_b) with angle-resolved exhaust in such a single-cylinder test engine (SCRE). For such a purpose, the P_b values around 1, 1, 1, 4 as well as 1,8 bar with conventional SCRE diesel combustion were considered. In addition, its influence for boost pressure (P_{in}) around 1.2 as well as 2.4 bar mostly on thermo-mechanical elements including its exhaust system is recorded at various P_b levels. Activity achievable mostly during the blow-down as well as during the displaced processes, including the exhaust process, is mostly computed. Notwithstanding P_{in} , with an improvement in P_b , the overall volume of exercise decreased similarly during the blowing process. This amount was determined for the fuel available exergy as well as exhaust exergy. Ultimately, whereas the research review emphasizes mostly on exergy of exhaust efficiency emission with diesel combustion throughout the SCRE, the combined approach, by the analysis of exhaust energy flows through multiple engines as well as combustion processes, can also be easily accepted in order to promote optimum WER exhaustion [10].

Because experimental testing methods usually entail running costs, quantitative approaches to generating empirical findings have become inevitable as a research method, or even sometimes remain the only feasible procedure. As in the sense of such topics, this article addresses a range of trend-setting simulation techniques of (1) quantitative, (2) regression and (3) artificial neural networking approaches as well as optimization methods of (1) surface response strategy, (2) Taguchi process and (3) genetic algorithms that have been commonly used in automotive research. The study proposes the incorporation of advanced statistical methods and new conventional machine learning algorithms with engine analysis in the process of extracting detailed functional models as an analytical method [11, 12, 13].

The advent of Diesel Particulate Filters (DPFs) as well as Selective Catalytic NO_x Reduction (SCR) systems has lowered the Particle Number (PN) and NO_x emissions to quite lower concentrations. Author has evaluated controlled as well as non-regulated Euro 6d-temp vehicle emissions both within the laboratories as well as on the road. As a result of regeneration, certain experimental PN and NO_x limits, while not sufficient, were exceeded during second evaluated situations. Even in the context of restoration,

on-road emissions were below the applicable non-exceeding requirements. Initial results of study indicate that, owing to the short time between regeneration, diesel vehicle studies will record emissions during regeneration activities [14].

Catalyzed diesel particulate filters (DPF) were already identified as multipurpose homogeneous catalysts. That purpose of such a project is to identify the low-complexity SCR-filter system that retains good stability. An elevated SCR-coated filter framework has been created as well as verified. The efficiency of its model has been defined throughout the original post. Research article helps to minimize complexity of the model. The intention is to achieve simulation cycles that could enable such an online control system architecture to be applied through the engine control unit. Two strategies for both the SCR-coated filter model order reduction (MOR) technique have been used for the "grey-box" methodology by balanced orthogonal decomposition (POD) as well as the "black box" methodology through the use of the artificial neural network (ANN) strategy. This POD system has also been shown to produce large MOR though retaining its significant level of reliability but with much less than 5% increase of simulation time. The ANN system provides a significant MOR with such a decrease of simulation time by orders of magnitude. That precision of both the ANN model becomes adequate with such a good generalization of its recent test results, however slightly lower than the POD process [15].

Although in the current review, both the performance and the exhaust emissions of this kind of single-cylinder direct-injection and air-cooled diesel engine using diethyl ether (DEE) diesel fuel mixtures have been calculated by artificial neural networks (ANN). The research experiment was conducted on pure diesel, including diesel-DEE blends for differing engine loads and speeds, for the acquisition of the requisite experiments, as well as the learning algorithm needed for the construction of its ANN model [16]. The ANN system was developed using 75% of its preliminary training findings. This performance, including its ANN model, was determined by multiplying the test results produced primarily by the remaining part of the planning phase. Such results mean that perhaps the ANN model could be used to estimate emissions and efficiency of low-power diesel engines [17, 18].

3. Identification Of Variables

The dependent or response variable for the system under consideration is back pressure on engine (P_b). There are many independent variables involved in this system. Out of all geometric features/ variables of Tested Diesel Particulate Filter design, engine operating parameters and environmental variables are considered in this work. Back pressure on engine (P_b) is considered as a dependent variable which depends upon number of plates used (n), number of perforations per unit area of the plate used (P_p), smaller diameter of cone (C_{d1}), larger diameter of cone (C_{d2}), axial length of cone (C_a), exhaust outlet diameter (E_o), thickness of plate (T_p), ambient pressure of air (P_o), density of ambient air (ρ) and acceleration due to gravity (g) [19, 20, 21].

Applying Buckingham's π -theorem, π terms are determined as given below. Mathematically, it can be stated as,

$$P_b = f_1(n, P_p, C_{d1}, C_{d2}, C_a, E_o, T_p, P_o, \rho, g) \dots\dots\dots (3.1)$$

Or it may be given as

$$f_1(P_b, n, P_p, C_{d1}, C_{d2}, C_a, E_o, T_p, P_o, \rho, g) = 0 \dots\dots\dots (3.2)$$

Thus, its total amount of variables, $n = 11$; No. of fundamentals dimensions, $m = 3$

Thus the amount of dimensionless π - terms can be given as $(n - m) = 11 - 3 = 8$

Therefore, seven π - terms has been established. As a consequence equation (II) becomes described below,

$$f_1(\pi_1, \pi_2, \pi_3, \pi_4, \pi_5, \pi_6, \pi_7, \pi_8) = 0 \dots\dots\dots(3.3)$$

Each π -term must comprise of $m + 1$ variable

Where m is 3 parameters and often referred as the repeat variables. Out of 10- variables, 3- variables must be identified as repeat variables; Taken P_b as dependent variable, which, therefore can not be taken

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for instance repeat variables. Out from the rest of the 10 variables (variables must include geometric properties, flow properties as well as fluid properties). 3-dimensions, Ca, g and ρ parameters are selected for instance repeat variables. Variables parameters itself must not create a dimensionless concept but should have fundamental dimension equivalent to m, i.e. 3 here. The dimension of Ca, g and ρ are taken as L, LT⁻², ML⁻³ and Therefore, there are three fundamental dimensions through Ca, g and ρ, they itself do not establish a dimensionless Pi terms group.

Every π –term are written as per the formula

$$\begin{aligned} \pi_1 &= C_a^{a1} * g^{b1} * \rho^{c1} * P_b ; \pi_2 = C_a^{a2} * g^{b2} * \rho^{c2} * n ; \\ \pi_3 &= C_a^{a3} * g^{b3} * \rho^{c3} * P_p ; \pi_4 = C_a^{a4} * g^{b4} * \rho^{c4} * C_{d1} \\ \pi_5 &= C_a^{a5} * g^{b5} * \rho^{c5} * C_{d2} ; \pi_6 = C_a^{a6} * g^{b6} * \rho^{c6} * E_o ; \\ \pi_7 &= C_a^{a7} * g^{b7} * \rho^{c7} * T_p \\ \pi_8 &= C_a^{a8} * g^{b8} * \rho^{c8} * P_o \end{aligned} \quad \dots\dots\dots (3.4)$$

The theory of dimensional homogeneity is used to define the π - terms. For determining the π – term, we can have

$$\pi_1 = M^0 * L^0 * T^0 = C_a^{a1} * g^{b1} * \rho^{c1} * P_b = (L)^{a1} (LT^{-2})^{b1} (ML^{-3})^{c1} ML^{-1} T^{-2}$$

Matching the powers of M, L, as well as T between both sides, thus will get power of L,

$$0 = a_1 + b_1 - 3c_1 - 1, a_1 = -1; \text{ Power of M, } 0 = c_1 + 1, c_1 = -1; \text{ Power of T, } 0 = -2b_1 - 2, b_1 = -1$$

Simply replace that values for a1, b1 as well as c1 throughout the formula (4),

$$\pi_1 = C_a^{-1} * g^{-1} * \rho^{-1} * P_b = \pi_1 = \frac{P_b}{C_a * g * \rho}$$

Or, as the components of this pi term concentrate mostly on variable, these are represented as

$$\pi_2 = C_a^{a2} * g^{b2} * \rho^{c2} * n = (L)^{a2} (LT^{-2})^{b2} (ML^{-3})^{c2} L$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a2, b2 and c2 throughout the formula (3.4),

$$\begin{aligned} \pi_2 &= C_a^{0} * g^{0} * \rho^{0} * n = \pi_2 = n \\ \pi_3 &= C_a^{a3} * g^{b3} * \rho^{c3} * P_p = (L)^{a3} (LT^{-2})^{b3} (ML^{-3})^{c3} L^{-2} \end{aligned}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a3, b3 and c3 throughout the formula (3.4),

$$\begin{aligned} \pi_3 &= C_a^{a2} * g^{0} * \rho^{0} * P_p = \pi_3 = P_p C_a^{-2} \\ \pi_4 &= C_a^{a4} * g^{b4} * \rho^{c4} * C_{d1} = (L)^{a4} (LT^{-2})^{b4} (ML^{-3})^{c4} L \end{aligned}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a4, b4 and c4 throughout the formula (3.4),

$$\begin{aligned} \pi_4 &= C_a^{-1} * g^{0} * \rho^{0} * C_{d1} = \pi_4 = \frac{C_{d1}}{C_a} \\ \pi_5 &= C_a^{a5} * g^{b5} * \rho^{c5} * C_{d2} = (L)^{a5} (LT^{-2})^{b5} (ML^{-3})^{c5} L \end{aligned}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a5, b5 and c5 throughout the formula (3.4),

$$\begin{aligned} \pi_5 &= C_a^{-1} * g^{0} * \rho^{0} * C_{d2} = \pi_5 = \frac{C_{d2}}{C_a} \\ \pi_6 &= C_a^{a6} * g^{b6} * \rho^{c6} * E_o = (L)^{a6} (LT^{-2})^{b6} (ML^{-3})^{c6} L \end{aligned}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a6, b6 and c6 throughout the formula (3.4),

$$\begin{aligned} \pi_6 &= C_a^{-1} * g^{0} * \rho^{0} * E_o = \pi_6 = \frac{E_o}{C_a} \\ \pi_7 &= C_a^{a7} * g^{b7} * \rho^{c7} * T_p = (L)^{a7} (LT^{-2})^{b7} (ML^{-3})^{c7} L \end{aligned}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a7, b7 and c7 throughout the formula (3.4),

$$\pi_7 = C_a^{-1} * g^{0} * \rho^{0} * T_p = \pi_7 = \frac{T_p}{C_a}$$

$$\pi_8 = C_a^{a8} * g^{b8} * \rho^{c8} * P_o = (L)^{a8} (LT^{-2})^{b8} (ML^{-3})^{c8} ML^{-1}T^{-2}$$

Matching that powers between M, L, as well as T from both sides, thus replace the quantities of a7, b7 and c7 throughout the formula (3.4),

$$\pi_8 = C_a^{-1} * g^{-1} * \rho^{-1} * P_o = \pi_8 = \frac{P_o}{C_a * g * \rho}$$

Substitute the value including its π -terms throughout the expression (3.3),

$$f_1 \left(\frac{P_b}{C_a * g * \rho}, n, P_p C_a^2, \frac{C_{d1}}{C_a}, \frac{C_{d2}}{C_a}, \frac{E_0}{C_a}, \frac{T_p}{C_a}, \frac{P_o}{C_a * g * \rho} \right) = 0$$

$$\text{or } \frac{P_b}{C_a * g * \rho} = f_1 \left(n, P_p C_a^2, \frac{C_{d1}}{C_a}, \frac{C_{d2}}{C_a}, \frac{E_0}{C_a}, \frac{T_p}{C_a}, \frac{P_o}{C_a * g * \rho} \right) = 0$$

Combining all Pi terms relating to Geometric features containing variables of Tested Diesel Particulate Filter design and environmental (Ambient pressure and density) variables in this category one independent Pi terms is formed. The Pi term π_w involves all geometric and environmental variables that are mandatory in the backpressure phenomenon analysis, i.e. this term has some positive value in all test points.

$$\pi_w = \left(\frac{n * P_p * C_{d1} * C_{d2} * E_0 * T_p * P_o}{C_a^3 * g * \rho} \right) \dots \dots \dots (3.5)$$

$$\pi_q = f(\pi_w) \text{ or } \left(\frac{P_b}{C_a * g * \rho} \right) = f \left(\frac{n * P_p * C_{d1} * C_{d2} * E_0 * T_p * P_o}{C_a^3 * g * \rho} \right) \dots \dots \dots (3.6)$$

4. Experimental Setup Used

Specifications of the diesel particulate filter (DPF):

- 1) Space velocity considered as: 50,000 hr⁻¹
- 2) Catalyst considered as: copper based.
- 3) Circular copper plates with 256 no. of holes per sq. cm.
- 4) Flange arrangement is made for dismantling along with perforated plates are shown in figure: 2.

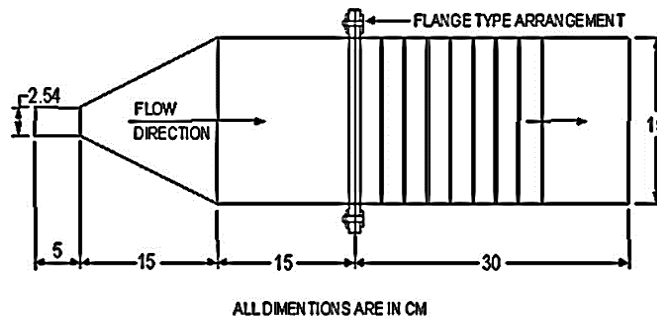


Figure.02: Schematic view of the Diesel Particulate Filter used for evaluation

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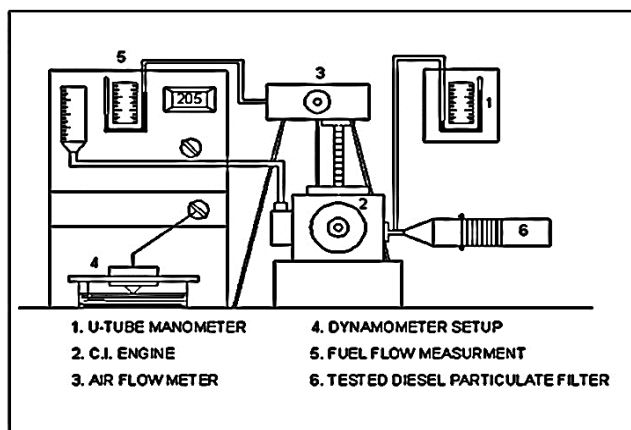


Figure.03: Schematic view of engine test set up

Engine Test Setup Specifications:

Sr. No.	Parameter	Specification
01	Make:	Single cylinder, Kirlosker Make, 4-stroke CI engine
02	Power Rated output:	5 H.P
03	Speed:	1500 Revolution Per Minute.
04	length of Stroke:	110 mm
05	Bore diameter:	80mm
06	Loading type:	Water - resistant form load change arrangements including copper element
07	Moment arm:	0.2 meter
08	Air box Orifice diameter:	25mm
09	Cd of orifice:	0.64

5. Experimental Procedure

Kirloskar make, 4-stroke, compression ignition engine as well as the diesel particulate filter to be evaluated are chosen for testing purpose. Throughout this analysis, during all the tests carried out, the mass flow rate of its The cold water of the engine jackets stayed unchanged at 0.1666 liters / s and also the engine speed of almost 1500 rpm to provide consistency in the comparison of some of the different variables. Perforated round copper plate configuration inside a DPF has been used as a test sample through a variation in backpressure. Further mostly while DPF test, new perforated plates & rings have been used every time [22, 23, 24].

The various variables are maintained as per expected range as per the input to DPF observed for changing performance parameters of the engine. The strategy to varying independent Pi term test plan is carried out, 100 numbers of perforated plates are prepared for backpressure modifications.in order to obtain various operating parameters. During the experiment, data on the test setup's independent and dependent parameters were obtained. That reasons for deviations or variations in the measurements may also be down to a lack of regulation when another variables are held at its intended levels rather than to a specific inexperience in the calculation of the quantity [25, 26]. Every measurement is obtained whenever setup of its engine reaches a steady state to eradicate errors. Findings reported just after calculation of the engine output variables and

afterwards the adjustment of the Observation - based data in the functional form and measurements for the modeling process shall be carried out [27, 28, 29].

6. Development of Observational Data Oriented Model

In the design of the experiment, 1- independent term π_i (i.e. Π_w) and another dependent term π_i (i.e. Π_q) were defined. These π_i terms are essential for modelling approach. The term Based π_i was to be the version of the current one independent term π_i . Data mostly on system's independent as well as dependent variables were collected mostly during test [30, 31, 32, 33, 34]. Each quantifiable independent as well as dependent π_i concept related to the entire model would be used. The entire relation was little more than a theoretical equations, mainly as a development tool for the analysis with backpressure development hypotheses.

For the dependent π_i term π_q , we have,

$$\pi_q = K * (\pi_w)^a \quad \text{----- (6.1)}$$

Having taken the log from both sides of the whole formula, we see that,

$$\log \pi_q = \log K + a * \log (\pi_w) \quad \text{----- (6.2)}$$

Let, $\log \pi_q = Z_2$, $\log K = K'$, $\log (\pi_w) = A$,

Therefore the second formula can be expressed as

$$Z_2 = K' + a * A \quad \text{----- (6.3)}$$

The standard formulas referring with equation 3 would be as follows

$$\sum Z_2 = nK' + a * \sum A ; \sum Z_2 * A = K' * \sum A + a * \sum A * A \quad \text{---- (6.4)}$$

Where n represents the total of trials and also the quantities between values,

With every sets of equations, several quantity in multipliers K' and a , then it was substituted with that quantity for several unknowns. (i.e. K' and a). Which quantity for variables with L.H.S. and even some multipliers with K' as well as " a " were defined throughout the ranges of formulations. After replacing certain quantities in the formula (6.4), we'll have a set with two equations that need to be solve respectively to obtain the quantities of K' and a .

The matrix method used to solve such formulas utilizing 'MATLAB' as follows.

Let, $A = 2 \times 2$ matrix of the multipliers of K' and a ; $B = 2 \times 1$ matrix of the terms at L.H.S. and

$C = 1 \times 2$ matrix of solutions or values of K' and a ;

$$\text{Then, } C = \text{inv}(A) * B \quad \text{----- (6.5)}$$

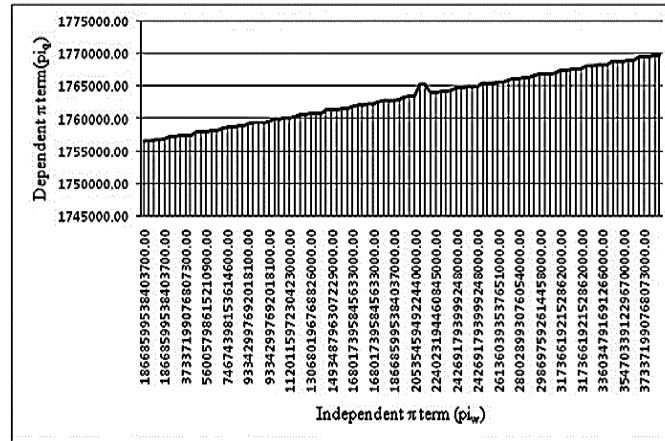
Provides the distinct values of K' and a , throughout this scenario as well as K' antilog, and then " a " will also be the solution of equation 1.

Thus, after substituting these values, the model would have been for π_q

$$\pi_q = 0.98995083 * (\pi_w)^{0.3636} \quad \text{----- (6.6)}$$

Therefore, according to a single dependent π_i term, researchers have developed a model depending on the evidence set for the backpressure generation model analysis of the phenomenon. Both based π_i terms being computed from such model values. The correlation coefficients between the observed and the measured values are considered as dependent π_i term shall be evaluated.

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Graph.1a: Variation between the dependent PI term (π_q) and the Independent PI term (π_w)

Qualitative Analysis of Data of Backpressure: The qualitative analysis of model of backpressure generation phenomenon analysis i.e. caused due to Tested Diesel Particulate Filter during experimentation is done. But, the model of backpressure generation phenomenon has only one independent pi term. From the graph: 1a, it is evident that, dependent pi term π_q is directly proportional to the independent pi term π_w . So for minimization of dependent pi term π_q value of independent pi term π_w must be minimized.

Model analysis for dependent pi term π_q :

$$\pi_q = 0.98995083 * (\pi_w)^{0.3636} \quad \text{-----(6.7)}$$

For this pi term, the deduced equation is expressed by

$$\pi_q = \left(\frac{P_b}{C_a * g * \rho} \right) \quad \text{-----(6.8)}$$

It has been shown again by formula (6.8), this is indeed a pi term model consisting of backpressure created mostly as a response variable by the evaluated Diesel Particulate Filter. The following basic hypotheses appear to be formed on the basis of these models.

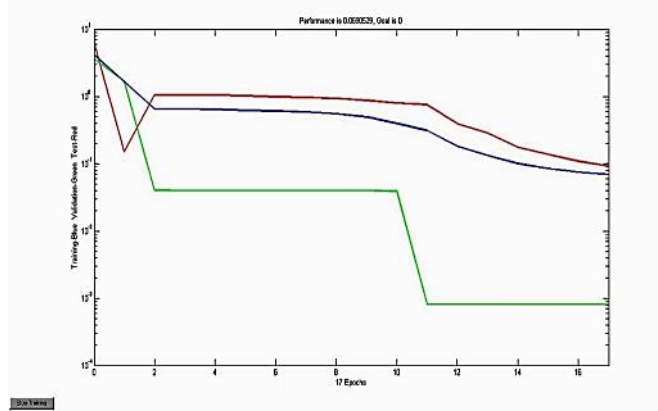
- i) π_w 's absolute index is 0.3636. The actual value is positive, meaning that π_w varies directly relative to π_q . Thus, within that equation, π_w seems to be the only affecting pi term.
- ii) That constant of such a model are equivalent to 0.98995083. Which shows that the sum influence of such a constant seems to be the compression or suppression of its actual value determined from such a pi term.
- iii) It's often noticed that even the magnitude of its numerator (viz. P_b) including its formula changes around 102998.586 and 103774.395, as well as the value including its denominator (viz. product of (C_a , g and ρ)) of its equation (3.4) remains unchanged at 7.205445. Such value during measurement are much more than one as well add the amplification impact to the model whenever we see the actions of the backpressure generating phenomena as regards to the various parameters.

The whole model is based on a study only of 20 sets for independent pi terms. Throughout the scenario of a backpressure phenomenon investigation, the values of only one separate pi term can still be adjusted. (viz. π_w). Most variable values remain unchanged throughout the experiment. Qualitatively, the measured values vary from its actual values obtained for the dependent π_q term. Thus, the actual behavior, including its formulated models, is also not evident from its current data collection. The objective of this experiment has been to establish a C.I. engine backpressure estimation framework. The phenomenon of backpressure development and its effect on the performance and emission characteristics of it's CI Engine. All models implemented were suitable for the unique configuration of the engine and also for the data set observed on their own, according to the recommended parameters.

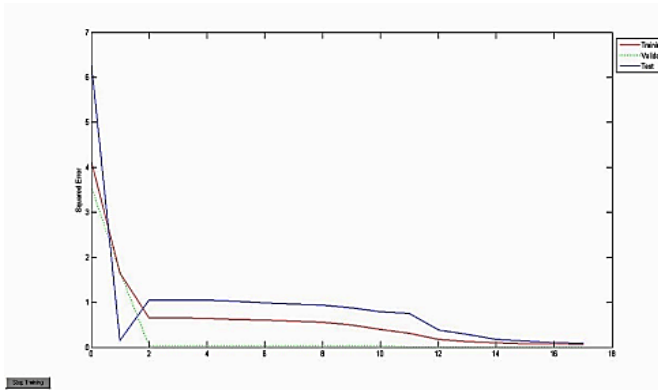
7. Effect of Modification on the dependent Pi Term π_q

Within that model, whenever a complete range of such a conditions that differ of 95 per cent is implemented the assessment of term piw, a variation of approximately 66.35 per cent emerges with in piq value (calculated using model).

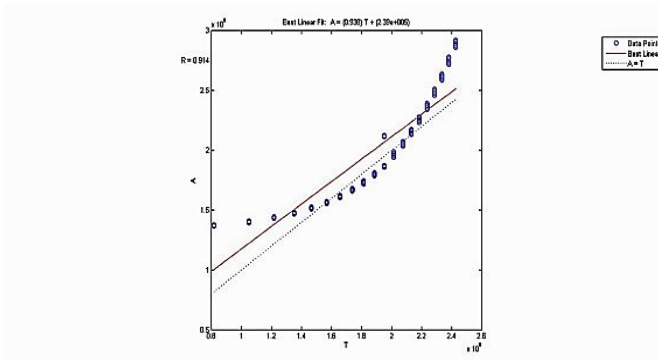
It has been shown that this model contains only one separate pi term piw and adjustments take place due to variations throughout this pi term.



Graph.2a: Analysis of results of the ANN model for Backpressure generation phenomenon analysis



Graph.3a: Analysis of results of the ANN model for Backpressure generation phenomenon analysis



Graph.2a: Correlation between Real Data against Computational Data in ANN Model with Backpressure Generation Phenomena Assessment

8. Procedure to Create the implementation strategy using ANN

For generation of ANN, various software / technology were developed. MATLAB as globally renowned

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approach has been adopted for creating the ANN only for complicated subject. That various phases identified throughout the ANN-formulating algorithm were as described beneath [35, 36, 37].

- i) A data recorded after performing its experiment is divided into two sections, i.e. Input data (independent) Pi terms data and output data (dependent) Pi terms' data. Independent and dependent data are stored in the form of test.txt as well as target.txt files.
- ii) Both input and output data can be accessed via the DLMREAD function.
- iii) Input and output data being normalized mostly during pre-processing step.
- iv) Standardized data is not correlated by principal component analysis. These are done using the PREPCA function. The inputs and outputs are then grouped into three groups. Testing, validation and training; standard procedure being to select an initial 75 per cent testing data collection, the very last 75 per cent validation outcomes as well as the central 50 per cent overlapping training records. These were achieved by utilizing a valid code.
- v) That standardized data, later it is processed throughout the validation as well as training testing environment.
- vi) While studying that data feed-forward back-propagation type of neural network form is selected.
- vii) ANN is trained through using standard data available for training. Calculation errors found in real as well as target data were estimated. That network would then be simulated. Errors within target (T) as well as actual data (A) were displayed through form of graph.
- viii) Uncorrelated output information is translated back to its original form with using POSTSTD feature.
- ix) Both regression analysis as well as interpretation were determined by means with basic features. Both values of its regression coefficient and also the regression line expression can be shown by 3- parameters (graphs: 2a, 2b and 2c) shown also for dependent term P_{iq} .

This even recommended that the models should be validated with experimental findings. The correlation revealed whether its accuracy between experimental and network outcomes was achieved by a typical absolute relative error to less than 2%. It is also thought that a well-trained ANN generates rapid and effective results, making it an excellent method for this type of thermal technical challenge in the laboratory investigation. [39, 40, 41, 42]

9. Results and Discussions

The findings of the trial were being converted into readable form. It has become necessary to analyze such data, which are supposed to lead to some logical conclusions.

Qualitative analysis of the data: To explain how well the actual phenomenon always develops upon its basis of a required relation of its independent Pi term (variables) the effort has been made as follows.

The actions of almost any model can indeed be qualitatively assessed by graphic illustration. This approach is being used here for the qualitative review. Engine operating performance variable mostly with the assist from its diesel particulate filter examine for experimental testing.

All readings of dependent Pi term have been calculated just after model has been defined using dependent Pi term. Observing slight difference in values of these terms, values are obtained by drawing the changes including its dependent Pi term mostly with independent Pi term. Therefore, for engine evaluation, there are 20 sets of independent Pi term readings as well as computed dependent Pi term readings. However, if we can draw its variability from a dependent Pi term via an independent Pi term, we get a graphical plot that could be seen in the Graph: 1a.

Quantitative analysis of the data: Statistical evaluation of the model indices, including its model indices, shows how well the phenomena is influenced by the interaction of different variables used in the model.

Now we are describing the effect of its independent Pi term indices mostly on dependent Pi term for analysis compared with results obtained after trials.

The model for evaluation does indeed have a constant (i.e. K) referred as constant of curve fitting. Inside an integrative manner, such constant commonly reflects the effect of a few of the variables that affect the phenomena and which was not necessarily changed mostly during analysis and therefore cannot be changed in a consistent type. Such independent influences are known to be extraneous variables with in philosophy of experimenting [1]. Whether situation is particular for proposed development. These variables must be evaluated and identified, such constants would then reflect the impact of fewer uncontrolled or extraneous variables inside an optimized manner. Throughout this case, the value of K is near unity mostly in experimental data-based model only for phenomenon of backpressure generation, and therefore effect of foreign variables would be almost zero.

Analysis of Performance of the Model: That model and the ANN have also been described mathematically. The values calculated only for independent Pi term mostly by mathematical model correlate really well with the values observed. The computed correlation coefficient value being -0.8874 throughout the experimental data-based modelling for backpressure development phenomena analysis for dependent Pi term (Piq). With Using excel, such factor is obtained. Through increasing the sets of its experiments, its correlation coefficient could further be strengthened. Its network developed utilizing MATLAB for such a model was effectively used to compute that dependent Pi term for such a specific set of independent Pi terms. Its value of the R squared error being 0.098, and well within the appropriate limits. The performance is normalized well after 9 iteration shown with graph: 2a. Also for dependent Pi term (Piq), that value of coefficient of regression is 0.937.

10. Conclusions

Through backpressure phenomenon analysis, another model has been developed for the dependent Pi term Piq. Which can be shown through the equation (6.6), model of a backpressure-containing Pi term created as a response variable according to the testing diesel particulate filter (P_b). Research approach is formulated for evaluation of its causes of backpressure rise using only a specific exhaust model mostly as case study with critical examination of backpressure phenomenon. Validation of such a work is often conducted with the aid of experimental data dependent model as well as ANN model for phenomena.

Computers have offered a simple ability to perform a number of complex problems using ANN methods. In addition to the advent of high-speed modern systems, the application of the ANN method could have advanced at a rather significant rate. It has been shown that the use of ANN is indeed an effective modelling method that has the ability to recognize different inter-relationships via input-output data. Even though no research indicates that, the effect of back pressure acting at the exhaust port, due to aftertreatment devices, on the performance and emission characteristics of the diesel engines used in the ANN approach have already been documented in this investigations. However the present research work examines the suitability of the same ANN methodology for estimating the requirements.

In order to minimize work of pumping, we have to take minimal backpressure as possible of we want to get the maximum performance of engine. The regeneration phenomenon within Diesel Particulate Filter seems to be of specific importance to the design and creation with Particulate Matter emissions testing initiatives. Backpressure seems to be exactly proportional to the pressure drop throughout the DPF. Therefore, in order to acquire a minimal back pressure mostly during various loading conditions of its engine, its appropriate design characteristics within each exhaust system part including its design are essential. Lowest possible back pressure for efficient usage of combustion energy under different operational conditions, without negatively impacting the engine output.

Nomenclature:

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Sr. No.	Parameters	Symbols used
01	Back pressure on engine	P_b
02	Number of plates used	n
03	Number of perforations per unit area of the plate used	P_p
04	Smaller diameter of cone	C_{d1}
05	Larger diameter of cone	C_{d2}
06	Axial length of cone	C_a
07	Exhaust outlet diameter	E_o
08	Thickness of plate	T_p
09	Ambient pressure of air	P_o
10	Density of ambient air	ρ
11	Acceleration due to gravity	g

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